

# Heat Transfer Studies on Plate Type Heat Exchanger Using Miscible And Immiscible Liquids

S.Ganapathy<sup>1</sup>, N.Tamilselvan<sup>2</sup>

<sup>1</sup>Department of Mechanical Engineering, Excel Engineering College, Komarapalayam,  
Corresponding Author: S.Ganapathy

**ABSTRACT:** Heat exchangers are used extensively and regularly in process and allied industries and are very important during design and operation. The choice of heat exchanger type directly affects the process performance and also influences plant size, plant layout, length of pipe runs, and the strength and size of supporting structures. Compact heat exchangers are characterized by having a comparatively large surface area to a given volume, when compared with traditional heat exchangers, in particular the shell and tube type. Two-phase (liquid-liquid) flow is frequently encountered in chemical process industries. In the present work the performance of plate type heat exchanger have been studied experimentally and comparison was made between the co-current and counter-current flow pattern with different cold side fluids that involve both immiscible and miscible systems. Hot water was taken as the hot side fluid for all the experimentation. The cold fluids used were water, kerosene-water system, acetic acid-water system of different compositions say 9%, 10%, 20% and 25% on volume basis in different flow rates. Also the flow rate of cold fluid was varied to find the effect of flow rate and composition on heat transfer coefficients, effectiveness and efficiencies. The hot fluid flow rate was kept constant at 1lpm and the cold fluid flow rate was varied from 12.5 to 42.5 lph. The above experiments indicate that the plate type heat exchanger performs well when operated under counter current flow pattern.

**KEYWORDS:** Water-Kerosene, Water – Acetic acid

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## I. INTRODUCTION

A heat exchanger is process equipment used for transferring heat from one fluid to another fluid through a separating wall. "When none of the fluid condenses or evaporates, the unit is called as Heat Exchanger." In this only the sensible heat transfers from the one fluid to another. They are widely used in petroleum refineries, chemical plants, petrochemical plants, natural gas processing, refrigeration, power plants, air conditioning and space heating. Heat exchangers could be classified in many different ways such as according to flow arrangements, number of fluids, surface compactness, process function, heat transfer mechanisms, type of fluids (gas-gas, gas-liquid, liquid-liquid, gas-two-phase, liquid-two-phase, etc.) and construction type (R.K. Shah *et.al.*,2000). There are several types of heat exchanger:

- Recuperative type, in which fluids exchange heat on either side of a dividing wall
- Regenerative type, in which hot and cold fluids occupy the same space containing a matrix of material that works alternatively as a sink or source for heat flow
- Evaporative type, such as cooling tower in which a liquid is cooled evaporatively in the same space as coolant.

### 1.1 OBJECTIVE OF THE PROPOSED WORK

The experimental work in plate type heat exchanger (Parallel and Counter flow) involved in the determination of outlet temperature of both cold and hot fluid for various flow rates. The water-water system, water-acetic acid system, and water-kerosene system at 9%, 10%, 20% and 25% composition on volume basis of kerosene and acetic acid, were used to determine the performance of plate type heat exchanger. i.e. overall heat transfer coefficient ( $U_0$ ), effectiveness ( $\epsilon$ ), cold side efficiency ( $\eta_c$ ) and hot side efficiency ( $\eta_h$ ).

## II. PLATE TYPE HEAT EXCHANGER

### 2.1 EXPERIMENTAL STUDIES

Experiments were carried out on a plate type heat exchanger with both parallel and counter flow patterns with miscible (acetic acid) and immiscible (Kerosene) systems in cold side and water in hot side

## 2.2 Experimental Set-up

The plate type heat exchanger under study consists of 5 plates and 2 end plates made of Aluminium of dimensions, 150 mm x 150 mm inner dimension and 250 mm x 250 mm outer dimension. The heat transfer plates are arranged to form a network of parallel or counter flow channels. The plate pack is mounted on upper and lower rails and compressed between two end frames using compression bolts. The hot fluid is hot water which is obtained from an electric geyser and it flows through one plate while the cold fluid is flowing through the other plate. The hot water flows always in one direction and the flow rate is controlled by means of a gate valve. The cold water can be admitted at one of the ends enabling the heat exchanger to run as a parallel flow apparatus or a counter flow apparatus by simple valve operation.



## 2.3 HEAT EXCHANGER SPECIFICATIONS

- No. of plates: 5 plates and 2 end plates.
- Size of the plate: outer diameter 215 mm x 215 mm inner diameter 150 mm x 150 mm.
- Material: Aluminium
- Gasket: silicon 6 nos.
- Temperature scanner 5 point 1 nos.
- Thermocouple 4 nos. 2 nos. for cold water inlet and outlet, 2 nos. for hot water inlet and outlet.
- Measuring tank 70mm in diameter length 500 mm 2nos.
- Stop watch- digital- 1 no.
- Geyser- single phase 230v-3kW-2 nos. total 6 nos. with main switch
- Insulation material: Asbestos

## 2.4 EXPERIMENTAL PROCEDURE

The experimental setup was operated in counter-current flow pattern by adjusting the valves. The flow of water on hot side was started. The flow rate of water entering the hot side of exchanger was adjusted with the help of a ball valve.

The geysers were switched ON and waited till temperature reaches a steady state. The temperature scanner was switched ON and inlet temperature of hot side ( $T_1$ ) was noted. The pumps were operated and the rotameters on cold side were adjusted to get the required flow rate of cold fluid with required composition (9% or 10% or 20% or 25% on the basis of volume).

The inlet temperature of cold fluid ( $t_1$ ) was noted from temperature scanner. The flow rates on the hot and cold side were unaltered until steady state is reached. When steady state is reached, outlet temperature of hot fluid ( $T_2$ ) and cold fluid ( $t_2$ ) were noted. The flow rate of cold fluid was varied and the above procedure was repeated. After completing a run with different flow rates for a particular solution at a particular composition, the procedure

was repeated for different compositions and different systems (Kerosene-Water, Acetic acid-Water). Similar procedure was followed for the co-current flow operation.

**2.5 EXPERIMENTAL OBSERVATION**

**2.5.1 Counter Current Flow Pattern**

**2.5.1.1 IMMISCIBLE SYSTEMS**

Volumetric flow rate of cold fluid lph	Volumetric flow rate of hot fluid lpm	Inlet Temperature °C		Outlet Temperature °C	
		Cold Fluid	Hot Fluid	Cold Fluid	Hot Fluid
<b>9% kerosene-water sytem</b>					
12.5	1	31	74	50	59
17.5	1	31	74	49	58
22.5	1	33	73	48	57
27.5	1	34	75	48	56
32.5	1	32	75	47	55
37.5	1	35	76	46	57
42.5	1	33	76	43	57
<b>10% kerosene- water system</b>					
12.5	1	29	76	50	67
17.5	1	30	77	51	65
22.5	1	31	76	51	64
27.5	1	33	75	48	63
32.5	1	35	76	51	63
37.5	1	36	75	52	64
42.5	1	37	75	51	65
<b>20% kerosene-water system</b>					
12.5	1	34	78	51	69
17.5	1	35	78	50	67
22.5	1	35	79	48	68
27.5	1	36	79	48	67
32.5	1	37	79	47	66
37.5	1	38	80	46	65
42.5	1	39	80	47	65
<b>25% kerosene- water system</b>					
12.5	1	31	73	48	59
17.5	1	33	76	47	67
22.5	1	34	75	50	64
27.5	1	33	74	48	62.9
32.5	1	35	72	48	61.9
37.5	1	32	77	46	59.9
42.5	1	36	78	45	58.9

**Table 2.1 Kerosene- water system**

**2.5.1.2 MISCIBLE SYSTEMS**

Volumetric flow rate of cold fluid lph	Volumetric flow rate of hot fluid lpm	Inlet Temperature °C		Outlet Temperature °C	
		Cold Fluid	Hot Fluid	Cold Fluid	Hot Fluid
<b>9% Acetic acid –water sytem</b>					
12.5	1	33	72	56.9	63
17.5	1	33	72	54.9	61
22.5	1	33	72	53.9	58
27.5	1	33	72	49.9	57
32.5	1	33	72	47.9	56
37.5	1	33	72	47.9	55
42.5	1	33	72	42.9	54
<b>10% Acetic acid – water system</b>					
12.5	1	33	72	53.9	63.9
17.5	1	33	72	51.9	62.9
22.5	1	33	72	50.9	58.9

27.5	1	33	72	50.9	57.9
32.5	1	33	72	49.9	55.9
37.5	1	33	72	48.9	52.9
42.5	1	33	72	46.9	54.9
20% Acetic acid –water system					
12.5	1	33	72	54	67
17.5	1	33	72	52	64
22.5	1	33	72	52	63
27.5	1	33	72	52	62
32.5	1	33	72	51	61
37.5	1	33	72	50	61
42.5	1	33	72	49	60
25% Acetic acid – water system					
12.5	1	33	72	60	64
17.5	1	33	72	58	63
22.5	1	33	72	58	62
27.5	1	33	72	57	62
32.5	1	33	72	54	61
37.5	1	33	72	55	61
42.5	1	33	72	54	60

**Table 2.2 Acetic acid- water system**

**2.5.2 Co-Current Pattern**

**2.5.2.1 IMMISCIBLE SYSTEMS**

Volumetric flow rate of cold fluid lph	Volumetric flow rate of hot fluid lpm	Inlet Temperature °C		Outlet Temperature °C	
		Cold Fluid	Hot Fluid	Cold Fluid	Hot Fluid
9% kerosene-water system					
12.5	1	31	74	45	65
17.5	1	31	74	46	63
22.5	1	33	73	45	61
27.5	1	34	75	45	63
32.5	1	32	75	43	63
37.5	1	35	76	44	64
42.5	1	33	76	43	64
10% kerosene- water system					
12.5	1	29	76	45	68
17.5	1	30	77	46	67
22.5	1	31	76	46	66
27.5	1	33	75	44	65
32.5	1	35	76	47	65
37.5	1	36	75	48	66
42.5	1	37	75	47	67
20% kerosene-water system					
12.5	1	34	78	48	71
17.5	1	35	78	48	68
22.5	1	35	79	47	68
27.5	1	36	79	48	67
32.5	1	37	79	47	66
37.5	1	38	80	46	65
42.5	1	39	80	47	65
25% kerosene- water system					
12.5	1	31	73	43	64
17.5	1	33	76	44	67
22.5	1	34	75	45	64
27.5	1	33	74	43	62
32.5	1	35	72	44	62
37.5	1	32	77	42	63
42.5	1	36	78	43	63

**Table 2.3 Kerosene – Water system**

**2.5.2.2 MISCIBLE SYSTEMS**

Volumetric flow rate of cold fluid lph	Volumetric flow rate of hot fluid lpm	Inlet Temperature °C		Outlet Temperature °C	
		Cold Fluid	Hot Fluid	Cold Fluid	Hot Fluid
9% Acetic acid -water sytem					
12.5	1	33	72	49	68
17.5	1	33	72	47	67
22.5	1	33	72	46	64
27.5	1	33	72	44	64
32.5	1	33	72	44	62
37.5	1	33	72	43	61
42.5	1	33	72	42	61
10% Acetic acid - water system					
12.5	1	33	72	48	66
17.5	1	33	72	47	65
22.5	1	33	72	46	63
27.5	1	33	72	44	62
32.5	1	33	72	43	61
37.5	1	33	72	43	61
42.5	1	33	72	42	60
20% Acetic acid -water system					
12.5	1	33	72	50	69
17.5	1	33	72	48	68
22.5	1	33	72	48	68
27.5	1	33	72	48	67
32.5	1	33	72	47	66
37.5	1	33	72	46	66
42.5	1	33	72	45	65
25% Acetic acid - water system					
12.5	1	33	72	50	67
17.5	1	33	72	48	66
22.5	1	33	72	48	65
27.5	1	33	72	47	64
32.5	1	33	72	46	62
37.5	1	33	72	45	61
42.5	1	33	72	44	60

**Table 2.4 Acetic acid – water system**

**2.6 CALCULATION METHOD**

The performance of the heat exchanger for different solutions of different concentrations is evaluated by calculating the overall heat transfer coefficient, NTU, exchanger effectiveness and efficiencies. The heat exchanger specifications are taken from the section 3.2.2. The calculation method is similar for co-current and counter-current operations. The calculation involves the following steps,

**1. Average Temperature:**

The average temperature of hot and cold fluid are calculated using the formula,

For cold fluid,  $t_{avg} = \frac{(t_1 + t_2)}{2}$

For hot fluid,  $T_{avg} = \frac{(T_1 + T_2)}{2}$

$t_1$  – Inlet temperature of cold fluid  
 $t_2$  – Outlet temperature of cold fluid  
 $T_1$  – Inlet temperature of hot fluid  
 $T_2$  – Outlet temperature of hot fluid

**2. Log Mean Temperature Difference:**

For counter-current flow,  $LMTD = \frac{((T_2 - t_1) - (T_1 - t_2))}{\ln \frac{(T_2 - t_1)}{(T_1 - t_2)}}$

For co-current flow,  $LMTD = \frac{((T_2 - t_2) - (T_1 - t_1))}{\ln \frac{(T_2 - t_2)}{(T_1 - t_1)}}$

**3. Fluid Properties:**

Fluid properties such as density, viscosity, specific heat capacity and thermal conductivity at average temperature of hot and cold fluid should be calculated. Properties of pure substance at average temperature are taken from Perry’s chemical engineering handbook. To find the properties of fluid mixture the following formulas are used,

$$\mu_{mix} = \mu_{pure}(x_1) + \mu_{water}(1 - x_1)$$

$$\rho_{mix} = \rho_{pure}(x_1) + \rho_{water}(1 - x_1)$$

$$c_{p\,mix} = c_{p\,pure}(x_1) + c_{p\,water}(1 - x_1)$$

$$k_{mix} = k_{pure}(x_1) + k_{water}(1 - x_1)$$

**4. Mass Flow Rate:**

The volumetric flow rate of hot and cold fluid are converted in to mass flow rate by multiplying it with their respective densities at average temperature.  
 Mass flow rate (m) = Volumetric flow rate x Density

**5. Dimensionless Numbers:**

Reynolds number, Nusselt number and Prandtl number for hot and cold fluids are calculated as follows;

Reynolds number =  $\frac{(D_e V \rho)}{\mu}$ .

Prandtl number =  $\frac{(c_p \mu)}{k}$ .

Nusselt number =  $\frac{(h D_e)}{k} = (0.28)(N_{Re})^{0.65}(N_{Pr})^{0.4}$ .

**6. Heat Transfer Coefficient:**

Individual heat transfer coefficient is calculated from respective Nusselt number. Overall heat transfer is found using the formula,

$$U_o = \frac{1}{\frac{1}{h_c} + \frac{x}{k} + \frac{1}{h_h}}$$

**7. Capacity Rate Ratio:**

Capacity rate = Mass flow rate (m) x Specific heat capacity (cp)

$$\text{Capacity rate ratio} = \frac{C_{\min}}{C_{\max}}$$

$$C_{\min} = m_c \times c_{pc}$$

$$C_{\max} = m_h \times c_{ph}$$

**8. Number of Transfer Units:**

$$NTU = \frac{(U_o \times A)}{C_{\min}}$$

**9. Efficiency:**

Efficiency for both hot and cold side are calculated using the formula,

$$\text{Cold side efficiency} = \eta_c = \frac{(m_c \times c_{pc} \times \Delta T)}{(m_c \times c_{pc} \times \Delta T_{\max})}$$

$$\text{Hot side efficiency} = \eta_h = \frac{(m_h \times c_{ph} \times \Delta T)}{(m_h \times c_{ph} \times \Delta T_{\max})}$$

**10. Effectiveness:**

$$\text{Effectiveness} = \frac{(NTU)}{(1 + NTU)} \times 100.$$

**III. PERFORMANCE ANALYSIS OF PLATE TYPE HEAT EXCHANGER**

**3.1 COUNTER CURRENT FLOW PATTERN**

**3.1.1 Immiscible Systems**

*Heat Transfer Studies On Plate Type Heat Exchanger Using Miscible And*

Table 3.1 Kerosene -water system

Cold Fluid				Hot Fluid (hot water)				Heat Transfer coefficient			Capacity Rate Ratio	No. of Transfer units	Effectiveness	Efficiency			
Fluid Temperature	Mass flow rate	Reynolds number	Nusselts number	Fluid temperature	Mass Flow rate	Reynolds number	Nusselts number	Cold side	Hot side	Overall				cold side	hot side		
K	Kg/sec			K	Kg/sec			W/m <sup>2</sup> K	W/m <sup>2</sup> K	W/m <sup>2</sup> K		NTU	%	%	%		
t <sub>1</sub>	t <sub>2</sub>	m <sub>c</sub>	N <sub>Re</sub>	N <sub>Nu</sub>	T <sub>1</sub>	T <sub>2</sub>	m <sub>h</sub>	N <sub>Re</sub>	N <sub>Nu</sub>	h <sub>c</sub>	h <sub>h</sub>	U <sub>c</sub>	R <sub>c</sub>	ε	η <sub>c</sub>	η <sub>h</sub>	
<b>9% Kerosene- Water System</b>																	
304.17	323.17	0.00336	12.4555	2.52857	347.17	332.17	0.01623	85.4683	7.4868	148.76	493.89	113.38	0.197022	2.47244	71.2018	44.186	34.884
304.17	322.17	0.0047	17.2782	3.1409	347.17	331.17	0.01624	84.8646	7.4765	184.61	492.83	132.99	0.275804	2.07129	67.4404	41.86	37.209
306.17	321.17	0.00604	22.4199	3.70512	346.17	330.17	0.01625	83.6641	7.4558	217.98	490.72	149.29	0.354402	1.8086	64.3951	37.5	40
307.17	321.17	0.00738	27.6555	4.22913	348.17	329.17	0.01624	84.2652	7.4662	249.05	491.78	163.35	0.433199	1.61937	61.8228	34.146	46.341
305.17	320.17	0.00873	31.7927	4.68808	348.17	328.17	0.01625	83.6641	7.4558	275.27	490.72	174.1	0.512044	1.45985	59.347	34.884	46.512
308.17	319.17	0.01007	37.3664	5.16421	349.17	330.17	0.01623	85.4683	7.4868	303.83	493.89	185.55	0.591065	1.34877	57.4246	26.829	46.341
306.17	316.17	0.01143	40.4245	5.54983	349.17	330.17	0.01623	85.4683	7.4868	324.86	493.89	193.19	0.670298	1.23831	55.3234	23.256	44.186
<b>10% Kerosene- Water System</b>																	
302.17	323.17	0.00335	12.3096	2.52427	349.17	340.17	0.01618	91.6062	7.5893	146.97	504.22	112.86	0.196026	2.47921	71.2579	44.681	19.149
303.17	324.17	0.00469	17.5538	3.15307	350.17	338.17	0.01619	90.9864	7.5791	183.94	503.21	133.39	0.274309	2.09349	67.6741	44.681	25.532
304.17	324.17	0.00603	22.7761	3.71946	349.17	337.17	0.0162	89.7494	7.5587	217.2	501.18	149.86	0.352481	1.82961	64.6595	44.444	26.667
306.17	321.17	0.00737	27.5845	4.22985	348.17	336.17	0.01621	88.5174	7.5382	246.76	499.11	163.15	0.430667	1.62948	61.9696	35.714	28.571
308.17	324.17	0.0087	34.1026	4.7584	349.17	336.17	0.0162	89.1327	7.5484	278.89	500.15	176.73	0.508771	1.49448	59.9114	39.024	31.707
309.17	325.17	0.01003	40.049	5.24112	348.17	337.17	0.0162	89.1327	7.5484	307.72	500.15	187.88	0.586891	1.37731	57.9357	41.026	28.205
310.17	324.17	0.01137	45.3889	5.68534	348.17	338.17	0.0162	89.7494	7.5587	333.8	501.18	197.46	0.665293	1.27719	56.0863	36.842	26.316
<b>20% Kerosene- Water System</b>																	
307.17	324.17	0.00326	13.9502	2.60983	351.17	342.17	0.01616	94.0871	7.63	139.85	508.15	108.79	0.18151	2.58342	72.0937	38.636	20.455
308.17	323.17	0.00457	19.5303	3.24785	351.17	340.17	0.01617	92.847	7.6096	174.04	506.21	128.29	0.253998	2.17605	68.5143	34.884	25.581
308.17	321.17	0.00588	24.6673	3.81009	352.17	341.17	0.01616	94.0871	7.63	203.8	508.15	143.93	0.326748	1.89854	65.4999	29.545	25
309.17	321.17	0.00718	30.4194	4.34898	352.17	340.17	0.01616	93.4673	7.6198	232.84	507.19	157.73	0.39925	1.70244	62.9964	27.907	27.907
310.17	320.17	0.00849	35.9502	4.84781	352.17	339.17	0.01617	92.847	7.6096	259.54	506.21	169.44	0.471733	1.54746	60.7453	23.81	30.952
311.17	319.17	0.00979	41.481	5.32037	353.17	338.17	0.01617	92.847	7.6096	284.84	506.21	179.87	0.544307	1.42369	58.7406	19.048	35.714
312.17	320.17	0.01109	47.8508	5.79256	353.17	338.17	0.01617	92.847	7.6096	310.67	506.21	189.84	0.616819	1.32593	57.0065	19.512	36.585
<b>25% Kerosene- Water System</b>																	
304.17	321.17	0.00323	13.7402	2.61369	346.17	332.17	0.01624	84.8646	7.4765	132.82	492.83	103.83	0.173828	2.56579	71.9557	40.476	33.333
306.17	320.17	0.00452	19.4125	3.25882	349.17	340.17	0.01618	91.6062	7.5893	165.77	504.22	123.62	0.243964	2.18208	68.574	32.558	20.93
307.17	323.17	0.0058	25.8709	3.86598	348.17	337.17	0.0162	89.1327	7.5484	197.38	500.15	140.07	0.313381	1.92302	65.7888	39.024	26.829
306.17	321.17	0.0071	30.783	4.37997	347.17	336.1	0.01621	87.8609	7.5272	223.01	498	152.31	0.382845	1.71085	63.1112	36.585	27
308.17	321.17	0.00839	37.0386	4.90069	345.17	335.1	0.01623	86.0316	7.4964	249.98	494.86	164.09	0.452148	1.55955	60.9306	35.135	27.216
305.17	319.17	0.00969	40.8442	5.3279	350.17	333.1	0.01621	87.8609	7.5272	270.49	498	173.06	0.52206	1.42554	58.7721	31.111	37.933
309.17	318.17	0.01097	47.5737	5.81243	351.17	332.1	0.01621	87.8609	7.5272	295.94	498	183.14	0.59167	1.33107	57.1012	21.429	45.405





3.2.2 Miscible Systems

Table 3.4 Acetic acid -water system

Cold Fluid					Hot Fluid (hot water)					Heat Transfer coefficient			Capacity Rate Ratio	No. of Transfer units	Effectiveness	Efficiency	
Fluid Temperature		Mass flow rate	Reynolds number	Nusselts number	Fluid temperature		Mass Flow rate	Reynolds number	Nusselts number	Cold side	Hot side	Overall				cold side	hot side
K		Kg/sec	$N_{Re}$	$N_{Nu}$	K		Kg/sec	$N_{Re}$	$N_{Nu}$	W/m <sup>2</sup> K	W/m <sup>2</sup> K	W/m <sup>2</sup> K	$R_c$	NTU	%	%	%
$t_1$	$t_2$	$m_c$	$N_{Re}$	$N_{Nu}$	$T_1$	$T_2$	$m_h$	$N_{Re}$	$N_{Nu}$	$h_c$	$h_h$	$U_o$	$R_c$	NTU	$\epsilon$	$\eta_c$	$\eta_h$
<b>9% Acetic acid – Water System</b>																	
306.17	322.17	0.00344	11.3631	2.49028	345.17	341.17	0.0162	89.7494	7.5587	147.32	501.18	112.91	0.20227	2.4021	70.6066	41.026	10.256
306.17	320.17	0.00482	15.6212	3.08778	345.17	340.17	0.0162	89.1327	7.5484	182.32	500.15	132.32	0.28322	2.01	66.7775	35.897	12.821
306.17	319.17	0.0062	19.9026	3.62913	345.17	337.17	0.01622	87.2919	7.5177	214.07	497.03	148.01	0.36395	1.7484	63.6151	33.333	20.513
306.17	317.17	0.00758	23.8877	4.11991	345.17	337.17	0.01622	87.2919	7.5177	242.53	497.03	161.07	0.44498	1.5563	60.8807	28.205	20.513
306.17	317.17	0.00896	28.231	4.59246	345.17	335.17	0.01623	86.0741	7.4971	270.35	494.94	172.63	0.52565	1.4114	58.5298	28.205	25.641
306.17	316.17	0.01034	32.2803	5.03113	345.17	334.17	0.01623	85.4683	7.4868	295.87	493.89	182.55	0.60649	1.2932	56.3937	25.641	28.205
306.17	315.17	0.01172	36.2548	5.44786	345.17	334.17	0.01623	85.4683	7.4868	320.05	493.89	191.48	0.68747	1.1967	54.4766	23.077	28.205
<b>10% Acetic acid – Water System</b>																	
306.17	321.17	0.00344	11.2062	2.48586	345.17	339.17	0.01621	88.5174	7.5382	145.76	499.11	111.89	0.20126	2.3914	70.5136	38.462	15.385
306.17	320.17	0.00482	15.5467	3.08795	345.17	338.17	0.01621	87.9038	7.528	180.9	498.08	131.42	0.28174	2.0059	66.7326	35.897	17.949
306.17	319.17	0.0062	19.8081	3.62934	345.17	336.17	0.01622	86.682	7.5074	212.4	495.99	147.11	0.36214	1.7462	63.5856	33.333	23.077
306.17	317.17	0.00758	23.7759	4.12019	345.17	335.17	0.01623	86.0741	7.4971	240.65	494.94	160.02	0.44266	1.5535	60.8382	28.205	25.641
306.17	316.17	0.00896	27.8465	4.58462	345.17	334.17	0.01623	85.4683	7.4868	267.5	493.89	171.34	0.52312	1.4073	58.4589	25.641	28.205
306.17	316.17	0.01034	32.1306	5.03152	345.17	334.17	0.01623	85.4683	7.4868	293.57	493.89	181.68	0.6036	1.2932	56.3926	25.641	28.205
306.17	315.17	0.01173	36.0883	5.44833	345.17	333.17	0.01624	84.8646	7.4765	317.57	492.83	190.43	0.68404	1.1958	54.459	23.077	30.769
<b>20% Acetic acid – Water System</b>																	
306.17	323.17	0.00346	10.8923	2.49942	345.17	342.17	0.01619	90.3674	7.5689	135.18	502.2	105.69	0.19167	2.3733	70.3553	43.59	7.6923
306.17	321.17	0.00484	14.9754	3.09895	345.17	341.17	0.0162	89.7494	7.5587	167.31	501.18	124.29	0.26836	1.9931	66.5896	38.462	10.256
306.17	321.17	0.00622	19.2541	3.64887	345.17	341.17	0.0162	89.7494	7.5587	197	501.18	139.96	0.34503	1.7456	63.5783	38.462	10.256
306.17	321.17	0.00761	23.5327	4.15725	345.17	340.17	0.0162	89.1327	7.5484	224.44	500.15	153.18	0.42161	1.5631	60.9854	38.462	12.821
306.17	320.17	0.00899	27.5639	4.62569	345.17	339.17	0.01621	88.5174	7.5382	249.5	499.11	164.35	0.49822	1.4188	58.6579	33.897	15.385
306.17	319.17	0.01038	31.5239	5.06754	345.17	339.17	0.01621	88.5174	7.5382	273.08	499.11	174.26	0.57495	1.3036	56.5903	33.333	15.385
306.17	318.17	0.01177	35.4149	5.48739	345.17	338.17	0.01621	87.9038	7.528	295.43	498.08	182.95	0.65156	1.2075	54.6999	30.769	17.949
<b>25% Acetic acid – Water System</b>																	
345.17	340.17	0.0162	89.1327	7.54843	306.17	323.17	0.00346	10.6511	2.5038	129.58	500.15	102.15	0.18673	2.3535	70.18	43.59	12.821
345.17	339.17	0.01621	88.5174	7.5382	306.17	321.17	0.00485	14.6445	3.1043	160.38	499.11	120.31	0.26143	1.9794	66.4367	38.462	15.385
345.17	338.17	0.01621	87.9038	7.52796	306.17	321.17	0.00624	18.8286	3.6552	188.84	498.08	135.57	0.33605	1.7348	63.4346	38.462	17.949
345.17	337.17	0.01622	87.2919	7.5177	306.17	320.17	0.00762	22.8095	4.1569	214.57	497.03	148.24	0.41068	1.5519	60.814	35.897	20.513
345.17	335.17	0.01623	86.0741	7.49714	306.17	319.17	0.00901	26.7215	4.6255	238.54	494.94	159.09	0.4852	1.4091	58.4899	33.333	25.641
345.17	334.17	0.01623	85.4683	7.48684	306.17	318.17	0.0104	30.567	5.0676	261.1	493.89	168.69	0.55979	1.2947	56.422	30.769	28.205
345.17	333.17	0.01624	84.8646	7.47651	306.17	317.17	0.01179	34.3482	5.4877	282.48	492.83	177.23	0.63437	1.2001	54.5472	28.205	30.769

IV. RESULTS AND DISCUSSION

5.1 COUNTER-CURRENT FLOW PATTERN

5.1.1 Effect Of  $N_{Re}$  (Cold) On Efficiency % (Cold)

5.1.1.1 Kerosene – Water System

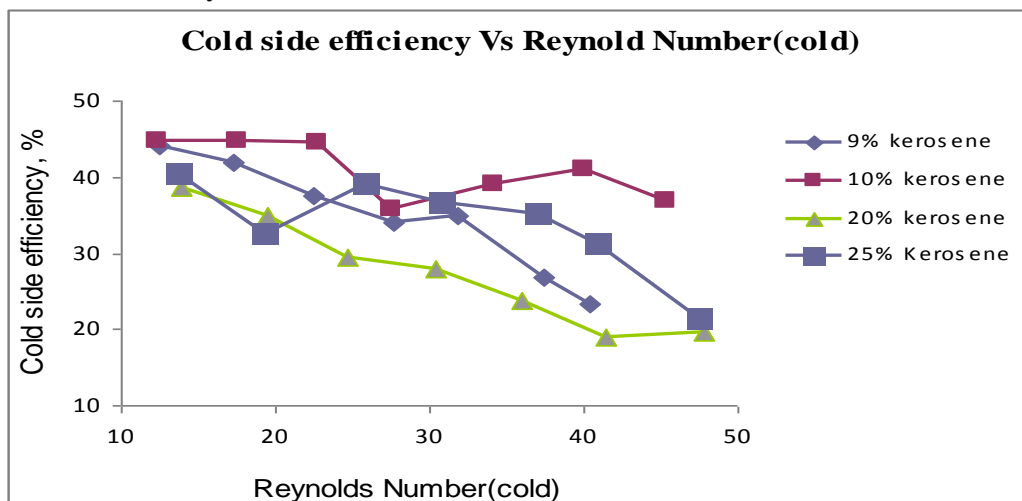
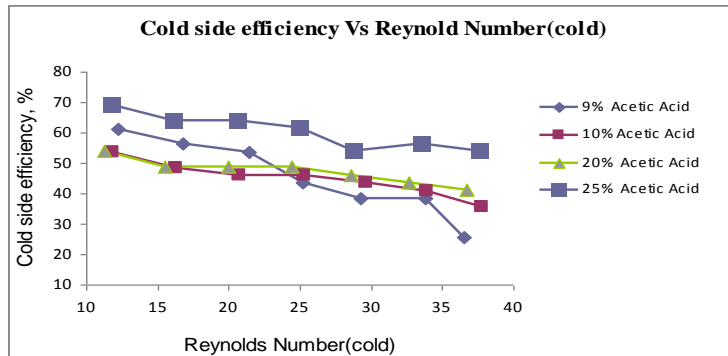


Fig. 5.1 Effect of cold side flow rate and composition on  $\eta_c$  of kerosene-water system

From the above graph it can be observed that, as  $N_{Re}$  (cold) increases the efficiency (cold) decreases. The graph also shows that as the composition of kerosene increases the efficiency of cold side increases and then decreases gradually for same flow rate of cold fluid.

### 5.1.2 Acetic acid-water System

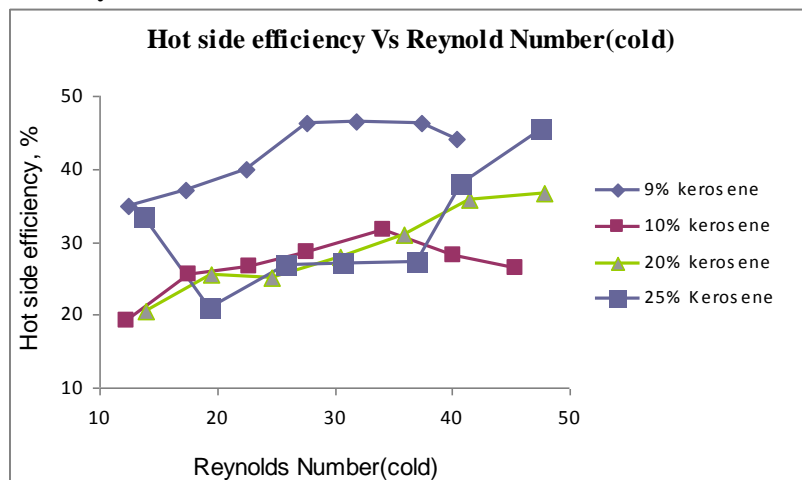


**Fig. 5.2 Effect of cold side flow rate and composition on  $\eta_c$  of acetic acid-water system**

It can be observed that, as flow rate of cold fluid increases the efficiency of cold side decreases. The graph also shows that as the composition of acetic acid increases the efficiency decreases for same flow rate of cold fluid. Maximum cold side efficiency is seen for 25% acetic acid -water system

### 5.1.2 Effect of $N_{Re}$ (Cold) On Hot Side Efficiency (%)

#### 5.1.2.1 Kerosene – Water System



**Fig. 5.3 Effect of cold side flow rate and composition on  $\eta_h$  of kerosene-water system**

From the graph Reynolds number of cold fluid Vs Hot side efficiency it can be observed that, as flow rate of cold fluid increases the efficiency of hot side gradually increases. When the flow rate on cold side is increased, heat transfer from hot to cold side is also increased. As the composition of kerosene increases from 9 % to 25 %, the efficiency of hot side decreases gradually for same flow rate of cold fluid.

#### 5.1.2.2 Acetic Acid – Water System

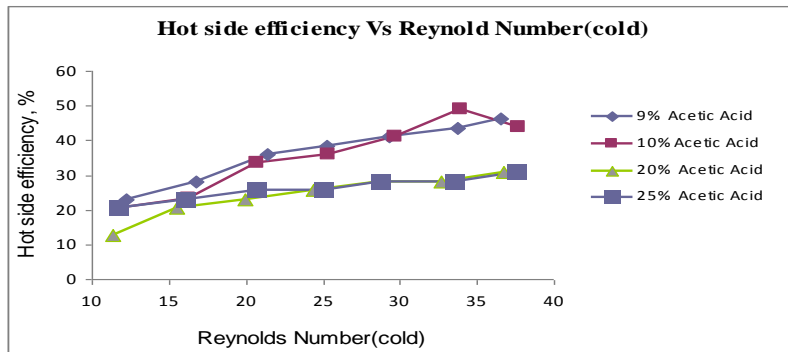


Fig. 5.4 Effect of cold side flow rate and composition on  $\eta_h$  of acetic acid-water system

From the graph it can be observed that, as flow rate of cold fluid increases the efficiency of hot side gradually increases. When the flow rate on cold side is increased, heat transfer from hot to cold side is also increased. As the composition of acetic acid increases, the efficiency of hot side decreases for same flow rate of cold fluid.

## 5.2 CO-CURRENT FLOW PATTERN

### 5.2.1 Effect of $N_{Re}$ (Cold) On Cold Side Efficiency (%)

#### 5.2.1.1 Kerosene – Water System

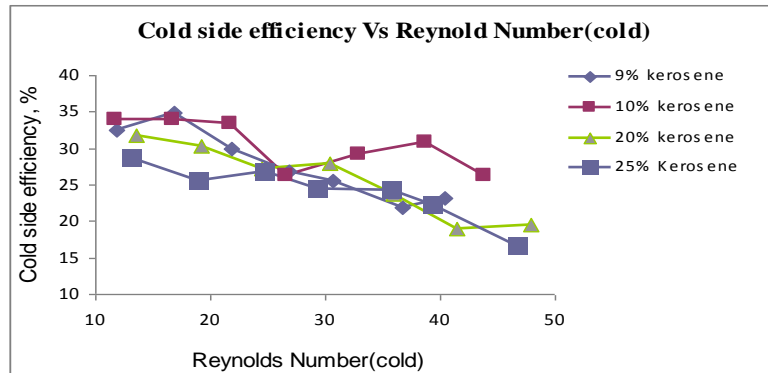


Fig. 5.5 Effect of cold side flow rate and composition on  $\eta_c$  of kerosene-water system

From the above graph it can be observed that, as  $N_{Re}$  (cold) increases the efficiency (cold) decreases. The graph also shows that as the composition of kerosene increases the efficiency of cold side decreases gradually for same flow rate of cold fluid.

#### 5.2.1.2 Acetic Acid – Water System

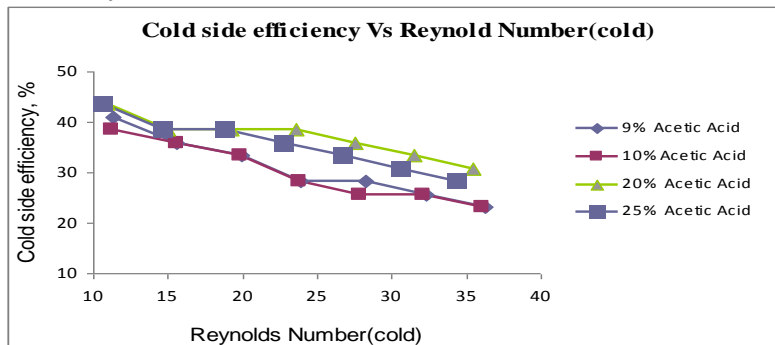


Fig. 5.6 Effect of cold side flow rate and composition on  $\eta_c$  of acetic acid-water system

The graph shows that, as  $N_{Re}$  (cold) increases the cold side efficiency decreases. As flow rate of cold side is increased the cold side efficiency decreases. The graph also shows that as the composition of acetic acid increases the efficiency of cold side increases gradually, for the same flow rate of cold fluid.

### 5.2.2 Effect of $N_{Re}$ (Cold) On Hot Side Efficiency (%)

#### 5.2.2.1 Kerosene – Water System

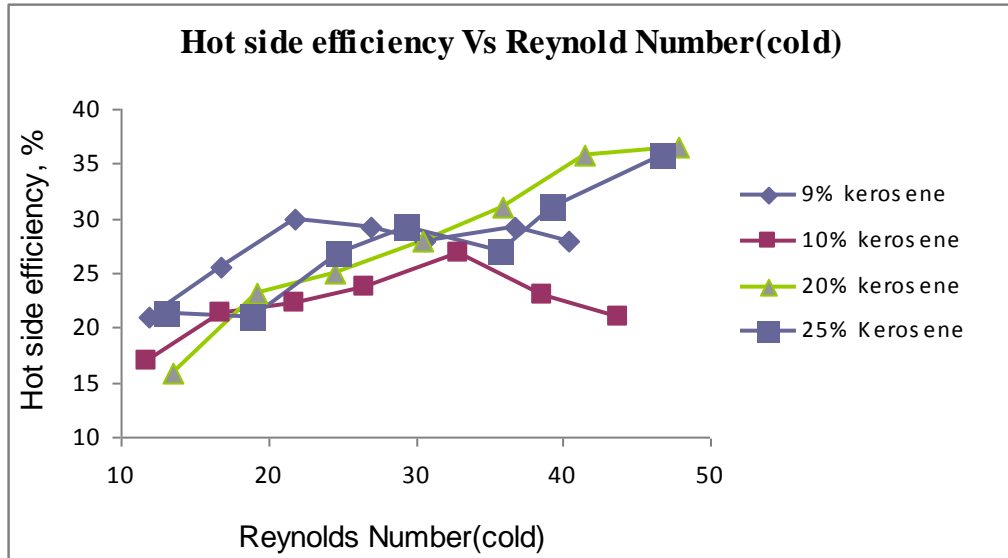


Fig. 5.7 Effect of cold side flow rate and composition on  $\eta_h$  of kerosene-water system

From the graph Reynolds number of cold fluid Vs Hot side efficiency it can be observed that, as flow rate of cold fluid increases the efficiency of hot side gradually increases. When the flow rate on cold side is increased, heat transfer from hot to cold side is also increased. As the composition of kerosene increases, the efficiency of hot side decreases at first and later it increases gradually for same flow rate of cold fluid.

#### 5.2.2.2 Acetic Acid – Water System

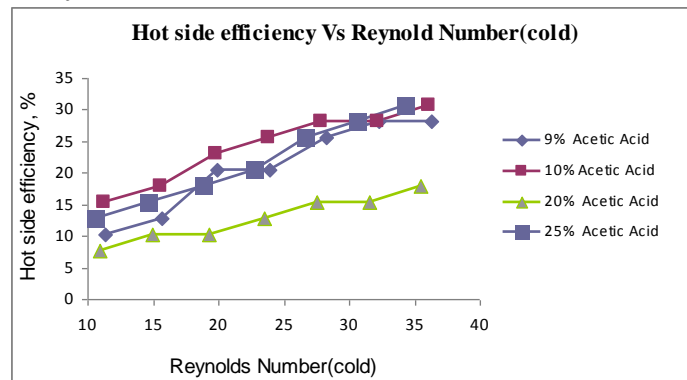


Fig. 5.8 Effect of cold side flow rate and composition on  $\eta_h$  of acetic acid-water system

From the graph Reynolds number of cold fluid Vs Hot side efficiency it can be observed that, as flow rate of cold fluid increases the efficiency of hot side gradually increases. When the flow rate on cold side is increased, heat transfer from hot to cold side is also increased. As the composition of acetic acid increases, the efficiency of hot side increases for same flow rate of cold fluid.

## V. CONCLUSION

Experiments were conducted on a Plate type heat exchanger with different flow rates and compositions of cold side fluid in both co-current and counter-current flow pattern. The effect of these parameters on heat transfer coefficient, efficiency and effectiveness were studied and comparison was made between co-current and counter-current flow operation.

It was found that as the flow rate of cold fluid increases, Reynolds number increases thereby increasing the cold side heat transfer coefficient. Consequently the overall heat transfer coefficient also increases with increase in flow rate of cold fluid for both counter-current and co-current flow pattern. The graphs also show that as the composition of cold fluid increases the  $N_{Nu}$  number decreases gradually for immiscible system, and increases for miscible systems. Therefore the cold side heat transfer coefficient and overall heat transfer coefficient decreases for immiscible systems and increases for miscible systems.

For both counter-current and co-current flow pattern as  $N_{Re}$  (cold) increased the  $N_{Nu}$  (hot) decreased thereby, decreasing the heat transfer coefficient of the hot side. As the flow rate on cold side increased the heat transfer coefficient on hot side ( $h_h$ ) decreased thereby decreasing the overall heat transfer coefficient but as the composition of cold fluid increases the heat transfer coefficient of hot side increases for the same flow rate of cold fluid

It was observed that the cold side efficiency and hot side efficiency for counter flow pattern is greater than the co-current flow pattern

From the results and discussion it can be concluded that, as flow rate of cold fluid increased the effectiveness gradually decreased. As the composition of cold fluid increased from 9% to 25%, the effectiveness of exchanger increased for immiscible systems and decreased for miscible systems for same flow rate of cold fluid for both counter current and co-current flow pattern.

Based on the overall results and discussion it was found that the overall heat transfer coefficient, hot side efficiency, cold side efficiency and effectiveness was greater for counter current flow pattern when compared to co-current flow pattern.

Therefore it can be concluded that the counter-current flow pattern was efficient and feasible compared to co-current flow pattern irrespective of the type of cold fluid used.

## APPENDIX I

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### MODEL CALCULATION

#### Counter-Current Flow Pattern

$$\text{Average temperature of cold fluid} = \frac{(t_1 + t_2)}{2} = \frac{(304.17 + 323.17)}{2} = 313.67 \text{ K}$$

$$\text{Average temperature of hot fluid} = \frac{(T_1 + T_2)}{2} = \frac{(347.17 + 332.17)}{2} = 339.67 \text{ K}$$

$$\begin{aligned} \text{Logarithmic mean temperature difference} &= \frac{((T_2 - t_2) - (T_1 - t_1))}{\ln \left( \frac{T_2 - t_2}{T_1 - t_1} \right)} \\ &= \frac{((347.17 - 304.17) - (332.17 - 323.17))}{\ln \left( \frac{347.17 - 304.17}{332.17 - 323.17} \right)} \\ &= 21.739 \end{aligned}$$

$$\begin{aligned} \text{Mass flow rate of cold fluid } m_c &= \text{Volumetric flow rate of cold fluid} \times \text{Density} \\ &= (12.5 \times 10^{-3} \times 966.7414) / (60 \times 60) \text{ Kg/sec} \\ &= 0.0033567 \text{ Kg/sec} \end{aligned}$$

$$\text{Mass flow rate of hot fluid } m_h = \text{Volumetric flow rate of hot fluid} \times \text{Density}$$

$$= (1 \times 10^{-3} \times 973.9417) / (60) \text{ Kg/sec}$$

$$= 0.016123 \text{ Kg/sec}$$

Mass velocity of cold fluid = Mass flow rate / Flow area  
 = 0.0033567 / 0.0045  
 = 0.74593 Kg/m<sup>2</sup>sec

Mass velocity of hot fluid = Mass flow rate / Flow area  
 = 0.016123 / 0.0045  
 = 3.582888 Kg/m<sup>2</sup>sec

**Dimensionless numbers for cold fluid,**

$$\text{Reynolds number } N_{Re} = \frac{(D_e V \rho)}{\mu} = \frac{(0.01 \times 0.74593)}{0.000599}$$

$$= 12.455$$

$$\text{Prandtl number } N_{Pr} = \frac{(c_p \mu)}{k} = \frac{(3995.857 \times 0.000599)}{0.588332}$$

$$= 4.067$$

$$\text{Nusselt number } N_{Nu} = (0.28)(N_{Re})^{0.65}(N_{Pr})^{0.4} = (0.28)(12.455)^{0.65}(4.067)^{0.4}$$

$$= 2.52871.$$

**Dimensionless number for hot fluid,**

$$\text{Reynolds number } N_{Re} = \frac{(D_e V \rho)}{\mu} = \frac{(0.01 \times 0.016123)}{0.000422}$$

$$= 85.46826$$

$$\text{Prandtl number } N_{Pr} = \frac{(c_p \mu)}{k} = \frac{(4194.033 \times 0.000422)}{0.659672}$$

$$= 2.683293$$

$$\text{Nusselt number } N_{Nu} = (0.28)(N_{Re})^{0.65}(N_{Pr})^{0.4} = (0.28)(85.46826)^{0.65}(2.6832)^{0.4}$$

$$= 7.486838$$

Cold side heat transfer coefficient  $h_c$  =  $(N_{Nu} \times k) / (D_e)$   
 =  $(2.528571 \times 0.588332) / (0.01)$   
 = 148.764 W/m<sup>2</sup>K

Hot side heat transfer coefficient  $h_h$  =  $(N_{Nu} \times k) / (D_e)$   
 =  $(7.486838 \times 0.659672) / (0.01)$   
 = 493.8858 W/m<sup>2</sup>K

Overall heat transfer coefficient  $U_o = \frac{1}{\frac{1}{h_c} + \frac{x}{k} + \frac{1}{h_h}}$

$$= \frac{1}{\frac{1}{148.764} + \frac{0.02}{273} + \frac{1}{493.8858}}$$

$$= 113.3777 \text{ W/m}^2\text{K}$$

Capacity rate ratio = Cmin / Cmax

$$C_{\min} = m_c x c_{pc} = 0.0033567 \times 3995.857 = 13.4129$$

$$C_{\max} = m_h x c_{ph} = 0.016123 \times 4194.033 = 67.6203$$

Capacity rate ratio = 13.4129 / 67.6203  
= 0.1984

$$\text{Number of transfer units } NTU = \frac{(U_o x A)}{C_{\min}}$$

$$= (113.3777 \times 0.2925) / (13.4129)$$

$$= 2.47244$$

$$\text{Cold side efficiency } \eta_c = \frac{(m_c x c_{pc} x \Delta T)}{(m_c x c_{pc} x \Delta T_{\max})} = \frac{(13.4129 x 19)}{(13.4129 x 43)} = 0.44186 = 44.186\%$$

$$\text{Hot side efficiency } \eta_h = \frac{(m_h x c_{ph} x \Delta T)}{(m_h x c_{ph} x \Delta T_{\max})} = \frac{(67.6203 x 15)}{(67.6203 x 43)} = 0.3488 = 34.88\%$$

$$\text{Effectiveness \%} = \frac{(NTU)}{(1 + NTU)} x 100$$

$$= \frac{(2.47244)}{(1 + 2.47244)} x 100 = 71.202 \%$$

---

## APPENDIX II

### MODEL CALCULATION

#### Co-Current Flow Pattern

$$\text{Average temperature of cold fluid} = \frac{(t_1 + t_2)}{2} = \frac{(304.17 + 318.17)}{2} = 311.17 \text{ K}$$

$$\text{Average temperature of hot fluid} = \frac{(T_1 + T_2)}{2} = \frac{(347.17 + 338.17)}{2} = 342.67 \text{ K}$$

$$\text{Logarithmic mean temperature difference} = \frac{((T_2 - t_2) - (T_1 - t_1))}{\ln \left( \frac{(T_2 - t_2)}{(T_1 - t_1)} \right)}$$



$$= \frac{((347.17 - 318.17) - (338.17 - 304.17))}{\ln \left( \frac{347.17 - 318.17}{338.17 - 304.17} \right)}$$

$$= 31.433$$

Mass flow rate of cold fluid  $m_c$  = Volumetric flow rate of cold fluid x Density  
 =  $(12.5 \times 10^{-3} \times 967.8842) / (60 \times 60)$  Kg/sec  
 = 0.003361 Kg/sec

Mass flow rate of hot fluid  $m_h$  = Volumetric flow rate of hot fluid x Density  
 =  $(1 \times 10^{-3} \times 972.4119) / (60)$  Kg/sec  
 = 0.016207 Kg/sec

Mass velocity of cold fluid = Mass flow rate / Flow area  
 =  $0.003361 / 0.0045$  Kg/m<sup>2</sup>sec  
 = 0.74689 Kg/m<sup>2</sup>sec

Mass velocity of hot fluid = Mass flow rate / Flow area  
 =  $0.016207 / 0.0045$  Kg/m<sup>2</sup>sec  
 = 3.60156 Kg/m<sup>2</sup>sec

**Dimensionless numbers for cold fluid,**

Reynolds number  $N_{Re}$  =  $\frac{(D_e V \rho)}{\mu}$  =  $\frac{(0.01 \times 0.003361)}{0.000628}$   
 = 11.8895

Prandtl number  $N_{Pr}$  =  $\frac{(c_p \mu)}{k}$  =  $\frac{(3993.66 \times 0.000628)}{0.585352}$   
 = 4.28556

Nusselt number  $N_{Nu} = (0.28)(N_{Re})^{0.65} (N_{Pr})^{0.4}$  =  $(0.28)(11.8895)^{0.65} (4.2856)^{0.4}$   
 = 2.50506

**Dimensionless number for hot fluid,**

Reynolds number  $N_{Re}$  =  $\frac{(D_e V \rho)}{\mu}$  =  $\frac{(0.010 \times 0.016207)}{0.000407}$   
 = 88.51743

Prandtl number  $N_{Pr}$  =  $\frac{(c_p \mu)}{k}$  =  $\frac{(4195.917 \times 0.000407)}{0.662113}$   
 = 2.5784

Nusselt number  $N_{Nu} = (0.28)(N_{Re})^{0.65} (N_{Pr})^{0.4}$  =  $(0.28)(88.51743)^{0.65} (2.5784)^{0.4}$   
 = 7.5382

Cold side heat transfer coefficient  $h_c$  =  $(N_{Nu} \times k) / (D_e)$   
 =  $(2.505061 \times 0.58535) / (0.01)$   
 = 146.6344 W/m<sup>2</sup>K

Hot side heat transfer coefficient  $h_h = (N_{Nu} \times k) / (D_e)$   
 $= (7.5382 \times 0.662113) / (0.01)$   
 $= 499.114 \text{ W/m}^2\text{K}$

Overall heat transfer coefficient  $U_o = \frac{1}{\frac{1}{h_c} + \frac{x}{k} + \frac{1}{h_h}}$   
 $= \frac{1}{\frac{1}{146.6344} + \frac{10^{-2}}{273} + \frac{1}{499.114}}$   
 $= 112.4038 \text{ W/m}^2\text{K}$

Capacity rate ratio =  $C_{\min} / C_{\max}$

$C_{\min} = m_c \times c_{pc} = 0.0033361 \times 3993.663 = 13.42154$

$C_{\max} = m_h \times c_{ph} = 0.016207 \times 4195.917 = 68.0027$

Capacity rate ratio =  $13.42154 / 68.0027$   
 $= 0.1974$

Number of transfer units  $NTU = \frac{(U_o \times A)}{C_{\min}}$   
 $= (112.4038 \times 0.2925) / (13.4215)$   
 $= 2.449654$

Cold side efficiency  $\eta_c = \frac{(m_c \times c_{pc} \times \Delta T)}{(m_c \times c_{pc} \times \Delta T_{\max})} = \frac{(13.42154 \times 14)}{(13.42154 \times 43)} = 0.3255814 = 32.558 \%$

Hot side efficiency  $\eta_h = \frac{(m_h \times c_{ph} \times \Delta T)}{(m_h \times c_{ph} \times \Delta T_{\max})} = \frac{(68.0027 \times 9)}{(68.0027 \times 43)} = 0.2093 = 20.93 \%$

Effectiveness % =  $\frac{(NTU)}{(1 + NTU)} \times 100$   
 $= \frac{(2.4497)}{(1 + 2.4497)} \times 100 = 71.011 \%$

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