

Effect of operating conditions on vehicle braking efficiency Part 2: Effect of adhesion coefficients

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ABSTRACT: *The purpose of this paper is to study the effect of operating conditions on vehicle braking efficiency. In order to investigate the effects, a longitudinal dynamic model of the vehicle is proposed during braking on a flat road surface. The indicators such as braking distance and braking time are selected to investigate the effect of vehicle operating conditions on braking efficiency. The investigation results indicate that the operating conditions such as adhesion coefficients of road surface, vehicle braking speed have a great effect on vehicle braking efficiency, especially when the vehicle brakes on the road surfaces with low adhesion coefficients, and vehicle braking high speeds.*

KEYWORDS: *Braking system, Braking force, Operating condition, Braking efficiency*

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I. INTRODUCTION

In recent years, Vietnam has been integrating deeply with the world economy, including the automobile industry. World-famous car manufacturers such as Toyota, Mazda, Honda, Hyundai, Mercedes... have appeared quite a lot in Vietnam, along with the development of Vietnamese branded car companies such as Vinfast dominate the domestic market as well as begin exporting and setting up armor installation facilities in several countries. In order to meet the development requirements of the automobile industry, road infrastructure to serve circulation is necessary. When the roads are good, the traffic speed of cars increases, the requirement for safety of movement is especially important. Like other countries in the world, Vietnam is facing an increasing number of traffic accidents, causing great damage to people and property. Of these, about 70% of accidents are due to subjective causes and the rest are due to the technical system..

Therefore, promoting research and development of safety systems on cars to improve vehicle reliability as well as safe operation of the vehicle is extremely necessary. One of the technical equipment systems that ensure vehicle safety is the braking system on cars, which plays a very important role in ensuring movement safety and increasing vehicle operating efficiency. A lot of research has focused on improving the effectiveness of braking systems, which are essential for operating vehicles in safe conditions for both the driver and other road users. Therefore, the braking system is placed under strict safety conditions. The efficiency and reliability required for braking systems over time have been addressed and continuously improved [1]. The process of vehicle braking is to create frictional force for the purpose of reducing the movement of the vehicle until the vehicle stops. To evaluate the braking system, four criteria are used to evaluate the braking efficiency, which are deceleration when braking, braking time, braking distance and braking force or specific braking force. Continuous improvement is needed to achieve a high-performance braking system that ensures safe and stable braking at any speed. Yin et al. [2] after analyzing the braking process and using testing methods, identified five main parameters that affect braking efficiency, which are the friction coefficient between the brake disc and brake pad, as well as between the The tires and road surface, the control system, the driver and the vehicle's mass affected the braking process. Continuous improvement is needed to achieve a high-performance braking system that ensures safe and stable braking at any speed. In addition to influencing factors such as the structure of the brake system, during braking, the kinetic energy of the vehicle will convert a part to generate heat. At that time, the temperature of the friction brake mechanism components increases, which can lead to deformation or cracking of the disc, decrease in the coefficient of friction between the brake pad and the brake disc, evaporation of brake fluid, abrasion of the working surface. ,... all lead to a decrease in the efficiency of the vehicle's brakes. The braking ability of brake discs was later studied under extreme conditions by Rashid [3] and Sharip [4] and they suggested that brake discs need to transfer heat more effectively (better thermal conductivity), with good friction, high frictional thermal resistance, light weight, mechanical and thermal shock resistance [5], and all these factors depend on the construction variant chosen. Therefore, the regulations that must be followed by braking systems change continuously, and their performance and reliability criteria continue to improve [5], along with the materials used, through simulations and tested over time, and in the field. Nikit Gupta et al [6]

studied, analyzed the heat distribution in different regions as well as changed the brake disc materials and used ANSYS software to analyze with different brake disc surfaces. Choi and Lee [7] used the finite element method to analyze the thermal expansion in the disc brake system, the process of analyzing the thermal expansion problem when connecting between the components of the disc brake structure when the temperature is high. caused by friction. Aleksander A. Yevtushenko et al. [8] also used the finite element method to find a solution for the thermodynamic balance of friction and wear for materials used to fabricate cheeks. disc brake. From there, determine the change in braking time, friction coefficient, braking torque, engine speed, average temperature in the area of brake pads in contact with brake discs and wear due to surface friction.

Therefore, to ensure the safe operation of the vehicle, one of the requirements is the quality of the braking process to ensure the highest efficiency. The quality of the braking process is expressed through the criteria such as the braking distance, the braking time,... and the directional stability of the vehicle when braking. The main goal of this paper is to investigate the effect of the braking initial speeds of vehicle with the different adhesion coefficients on braking efficiency, a longitudinal dynamic model of the vehicle is proposed during braking on a flat road surface. The indicators such as braking distance and braking time are selected to investigate the effects.

II. VEHICLE LONGITUDINAL DYNAMIC MODEL

The process of vehicle braking is to create frictional force for the purpose of reducing the movement of the vehicle until the vehicle stops. To investigate vehicle braking efficiency, a longitudinal dynamic model of vehicle is established when the vehicle brakes on a flat road surface, as shown in Figure 1.

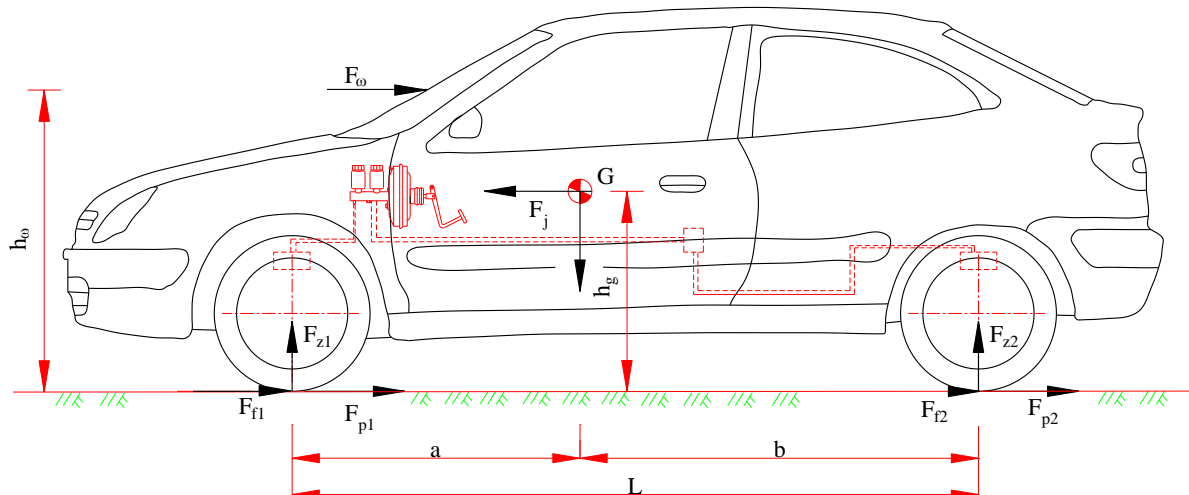


Fig 1. Dynamic model of the vehicle when braking

Interpretation of symbols on Figure 1, G is the vehicle weight; F_{p1} , F_{p2} are the braking forces at the front and rear wheels, respectively; F_0 is the air resistance force; F_j is the inertia force during braking; F_{z1} , F_{z2} are the vertical reaction forces of road surface on the wheels, respectively; a , b , and L are the distances, respectively and h_0 , h_g are the heights, respectively.

Based on Figure 1, total longitudinal braking force could be determined by the following formula

$$F_p = F_{p1} + F_{p2} \quad (1)$$

The total longitudinal braking maximum force is obtained as below

$$F_p = \varphi \cdot G \quad (2)$$

where, φ is adhesion coefficient of road surface.

Total rolling resistance force of vehicle could be determined by the following formula

$$F_f = F_{f1} + F_{f2} = f \cdot G \quad (3)$$

where, f is rolling resistance coefficient.

The air resistance force could be determined by the following formula

$$F_0 = K F v^2 \quad (4)$$

where, K is the coefficient of air resistance; F is the front bumper area of the vehicle; v is the relative velocity of movement between vehicle and longitudinal wind during braking

The force of inertia could be determined by the following formula

$$F_j = \delta_i \frac{G}{g} \frac{dv}{dt} \quad (5)$$

where, δ_i is the coefficient taking into account the influence of the rotating mass on the vehicle; t is the braking time; g is the acceleration due to gravity.

The longitudinal force balance equation of vehicle could be determined by the following formula

$$F_j = F_p + F_w + F_f \quad (6)$$

If the values of forces F_f , F_w , F_i are small, it could be neglected. Eq. (6) could become below

$$F_j = F_p \quad (7)$$

Combining Eq. (2) with Eq.(5), Eq. (6) could become below

$$\delta_i \frac{G}{g} \cdot \frac{dv}{dt} = \varphi \cdot G \quad (8)$$

From Eq.(8), the deceleration of acceleration during braking could be determined by the following

$$J_p = \frac{dv}{dt} = \frac{\varphi \cdot g}{\delta_i} \quad (9)$$

From Eq.(9), the braking time is determined by Eq.(11)

$$t_p = \int dt = \int_{v_2}^{v_1} \frac{\delta_i}{\varphi \cdot g} dv \quad (10)$$

$$\Rightarrow t_p = \frac{\delta_i}{\varphi \cdot g} (v_1 - v_2) \quad (11)$$

where, v_1 is the initial speed of the braking process, v_2 is the speed of the end of the braking process.

The braking time when the vehicle comes to a complete stop, $v_2=0$, Eq. (11) could become below

$$t_p = \frac{\delta_i}{\varphi \cdot g} v_1 \quad (12)$$

From Eq.(9), multiplying both sides of an equation by ds

$$\frac{dv}{dt} ds = \frac{\varphi \cdot g}{\delta} ds \Rightarrow v dv = \frac{\varphi \cdot g}{\delta} ds$$

$$\Rightarrow ds = \frac{\delta}{\varphi \cdot g} v dv \quad (13)$$

The braking distance is determined by Eq.(13).

$$s_p = \int ds = \int_{v_2}^{v_1} \frac{\delta}{\varphi \cdot g} v dv = \frac{\delta}{2\varphi \cdot g} (v_1^2 - v_2^2) \quad (14)$$

The braking time when the vehicle comes to a complete stop, $v_2=0$, Eq. (14) could become below

$$s_p = \frac{\delta}{2\varphi \cdot g} v_1^2 \quad (15)$$

From the above expression, we see that the minimum braking distance also depends on the speed of the vehicle, depends on adhesion coefficients and the factor taking into account the influence of the rotating masses, so the braking distance will be decrease when disengaging the clutch and then braking. The coefficient of traction depends on the load of the vehicle. As the load increases, the coefficient of traction will decrease, then the braking distance will increase. Therefore, the braking distance of different types will be different even though the initial braking speed is the same. We consider the effect of the initial speed when braking at different adhesion coefficients on the minimum braking distance.

III. RESULTS AND DISCUSSION

In order to investigate the effect of vehicle operating conditions on braking efficiency, the different adhesion coefficients of road $\varphi=[0.3 \ 0.4 \ 0.5 \ 0.6 \ 0.7 \ 0.8]$ with the braking initial speeds of vehicle such as $v_1=[2.78 \ 5.55 \ 8.33 \ 11.11 \ 13.89 \ 16.67 \ 19.4]$ m/s is selected to investigate the its effect on braking efficiency. The effect of different adhesion coefficients of road with the braking initial speeds of vehicle on braking time is shown Figure 2.

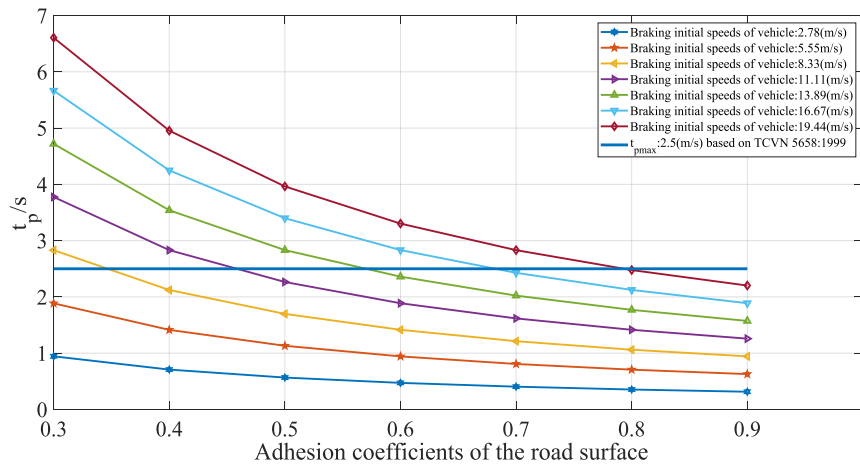


Fig 2. The effect of adhesion coefficients of road with the different braking initial speeds on the braking time

From the results on Figure 2, we show that the value of the different adhesion coefficients of road decreases, the braking time increases very quickly, especially, when low adhesion coefficients of the road surface $\varphi \leq 0.4$ and the vehicle brakes at the the high braking speeds $v_1 \geq 11.11$ m/s. Thus, in order to achieve the minimum braking time on roads at low adhesion coefficients, it is necessary to reduce vehicle speed when braking. The limited braking time according to Vietnamese standard TCVN 5658: 1999 [9], $t_p \leq 2.5$ s, the range of values $t_p \geq 2.5$ s, the operating conditions do not satisfy good braking performance.

The effect of the different adhesion coefficients on braking distance with the braking initial speeds of vehicle is shown Figure 3.

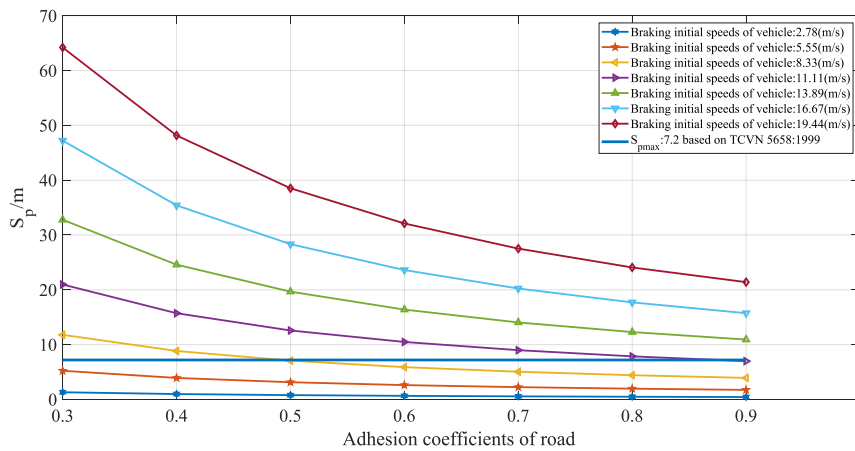


Fig 3. Effect of the different adhesion coefficients on braking distance with the braking initial speeds of vehicle

From the results on Figure 3, we show that the value of the different adhesion coefficients of road decreases, the braking distance increases very quickly, especially, when the vehicle brakes at the low adhesion coefficients of the road surface $\varphi \leq 0.4$ and the high braking speeds $v_1 \geq 8.33$ m/s. Thus, in order to achieve the minimum braking distance on roads at low adhesion coefficients, it is necessary to reduce vehicle speed when braking. The limited braking distance according to Vietnamese standard TCVN 5658: 1999 [9], $s_p \leq 7.2$ m, the range of values $s_p \geq 7.2$ m, the operating conditions do not satisfy good braking performance.

IV. CONCLUSION

In this study, in order to investigate the effect of the different adhesion coefficients on braking efficiency with the braking initial speeds of vehicle. A longitudinal dynamic model of vehicle was established when the vehicle brakes on a flat road surface. Some conclusions drawn from the investigation results: (i) The value of the different adhesion coefficients of road decreases, the braking time increases very quickly; (ii) The value of the different adhesion coefficients of road decreases, the braking distance increases very quickly, especially when the vehicle brakes with the braking high speeds on the road surfaces with low adhesion

coefficients. Therefore, in order to ensure effective braking of vehicles during traffic on different types of roads and loads, there should be appropriate maximum speed regulations.

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